## Drive selection <br> for linear drive units with toothed belt drive

Feed force
$\mathrm{F}_{\mathrm{X}}$ [N]

$$
\mathbf{F}_{\mathbf{x}}=\mathbf{m} \cdot \mathbf{g} \cdot \mu
$$

Acceleration force $\mathrm{F}_{\mathrm{a}}$ [N]

$$
\mathrm{F}_{\mathrm{a}}=\mathrm{m} \cdot \mathrm{a}
$$

In vertical applications, the mass acceleration a must be added to the acceleration due to gravity $\mathrm{g}\left[9.81 \mathrm{~m} / \mathrm{s}^{2}\right]$.

Power from torque and rotational speed [kW]

$$
P=\frac{M_{A} \cdot n_{\max } \cdot 2 \cdot \pi}{60 \cdot 1000}
$$

|  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
|  |  |  |  |  |
| $\begin{array}{ll} \mathrm{m} & =\text { Mass to be tranported }[\mathrm{kg}]^{11} \\ \mathrm{a} & =\text { Acceleration }\left[\mathrm{m} / \mathrm{s}^{2}\right] \\ \mathrm{d}_{0} & =\text { Effective diam. of pulley }[\mathrm{mm}]^{2)} \end{array}$ |  |  |  |  |
| $P$ $=$ Power $[k W]$ <br> $L$ $=$ WIESEL ${ }^{\circledR}$ length $[m m]$ <br> $J_{\text {syn }}$ $=$ Idle torque of pulley $\left[k g m^{2}\right]$ <br> $\mathrm{n}_{\max }$ $=$ Maximum rotational speed [rpm] <br> $\mu$ $=$ Friction factor |  |  |  |  |
| L $=$ WIESEL ${ }^{\oplus}$ length [mm] <br> $\mathrm{J}_{\text {syn }}$ $=$ Idle torque of pulley $\left[\mathrm{kgm}^{2}\right]$ <br> $\mathrm{n}_{\max }$ $=$ Maximum rotational speed [rpm] <br> $\mu$ $=$ Friction factor |  |  |  |  |
| $\begin{aligned} & J_{\text {syn }}=\text { Idle torque of pulley [kgm²] } \\ & \mathrm{n}_{\max }=\text { Maximum rotational speed [rpm] } \end{aligned}$ |  |  |  |  |
|  |  |  |  |  |
| $\mu=$ Friction factor |  |  |  |  |

## Calculating the drive moment $\mathrm{M}_{\mathrm{A}}$ [ Nm ]

The required drive moment is composed of the "load moment", the "acceleration moment" and the "idle torque".


| Type | $\mu$ | $J_{\text {syn }}\left[\mathrm{kgm}^{2}\right]$ | spec. mass tooth belt [kg/m] | Type | $\mu$ | $J_{\text {syn }}\left[\mathrm{kgm}^{2}\right]$ | Spec. mass tooth belt [kg/m] |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| WH40 | 0.05 | 8.800 E-06 | 0.032 | WHZ50 | 0.1 | 6.906E-05 | 0.055 |
| WH50 | 0.1 | 1.928 E-05 | 0.055 | WHZ80 | 0.1 | 5.026E-04 | 0.114 |
| WH80 | 0.1 | 2.473 E-04 | 0.210 | WM60 ZRT | 0.1 | $2.127 \mathrm{E}-05$ | 0.074 |
| WH120 | 0.1 | $1.004 \mathrm{E}-03$ | 0.340 | WM80 ZRT | 0.1 | $1.115 \mathrm{E}-04$ | 0.158 |
|  |  |  |  | MLSH60 ZRT | 0.1 | 4.604E-05 | 0.114 |

[^0]
[^0]:    1) Total mass $m=$ mass to be moved + mass of power bridge ${ }^{3)}+$ mass of toothed belt

    Mass of toothed belt = spec. mass of tooth belt $\left[\mathrm{kg} / \mathrm{ml} \cdot 2^{4)}\right.$. WIESEL*-lenght [mm]
    2) Values for the respective effective diametres, see at corresponding mechanical linear units.
    ${ }^{3)}$ For Z-axis moved dead mass to be taken into account.
    4) To replace by 1 at Z-Axis

